

The **Total Capital Investment** of a system (**TCI**) is only marginally composed of the **Purchase Equipment Cost** (**PEC**), which typically is between 15 and 40% of the Total Capital Investment (**TCI**).

In general:

$$TCI = FCI + AC = DC + IC + AC$$

Where:

FCI = **Fixed Capital Investment** (land purchase, buildings, purchase and installation of equipment,...)

AC = Additional Costs

DC = Direct Costs (equipment, buildings and all associated permanent structures)

IC = Indirect Costs (services and non-permanent structures)





The first step is certainly the evaluation of the **Purchase Equipment Cost (PEC)**. For a company it is often possible to get accurate estimates from previous projects, or to ask for a quotation.

Engineering Companies are usually able to evaluate PEC but are often reluctant to provide this info. Some softwares provide cost estimates for certain categories of equipment (e.g. Aspen Plus or Hysis for heat exchangers).

Textbooks in Chemical Engineering provide PEC evaluation charts. The estimates are often not accurate. In any case it is generally necessary to correct the chart cost data taking into account several variables, such as: design conditions (pressure, temperature,...), materials, and first of all **size of the equipment**.



Textbooks containing useful cost evaluation charts:

Garrett (Springer), **Peters-Timmerhaus** (McGraw-Hill), **Turton** (Prentice-Hall).

Turton book offers a popular spreadsheet (CapCost.xls).

Peters-Timmerhaus textbook offers a PEC calculation tool on the Web site: http://www.mhhe.com/engcs/chemical/peters/data/).

The estimates of spreadsheets or web sites are often not accurate.

Cost data from the web are useful for a check at power plant level but no disaggregation is present:

http://nyethermodynamics.com/trader/kwprice.htm

https://www.eia.gov/todayinenergy/detail.php?id=26532



Estimates of Purchased-Equipment Cost

The purchased-equipment cost (C_B) can be obtained through:

- vendors' quotations
- cost estimates from past purchase orders
- quotations from experienced professional cost estimators
- cost databases maintained by engineering companies
- commercial computer programs
- estimating charts (module method)

The module cost
$$C_M = C_B f_d f_m f_T f_p f_{BM}$$

or $C_M = C_B \left[(f_d + f_T + f_p) f_m + f_{BM} - 1 \right]$

where

 f_d – design-type factor; f_m – material factor; f_T – temperature factor; f_p – pressure factor; f_{BM} – bare module factors





EQUIPMENT COST ESTIMATES 287

Heat Exchangers: Spiral, Plate and Frame 304 stainless steel; no insulation



PEC Chart

PEC - HE

università degli studi FIRENZE

DIPARTIMENTO DI INGEGNERIA INDUSTRIALE

Heat Exchangers

Cost referred to surface



Table 2.3	Estimated	Costs of	Major	Equipment	(2008)	US\$)

Equipment Item	Estimated Cost, US\$/kW net
Pulverized Coal Boiler, subcritical, 325-MW gross	300
Pulverized Coal Boiler, subcritical, 540-MW gross	270
Pulverized Coal Boiler, supercritical, 860-MW gross	250
Steam Turbine, subcritical, 325-MW gross	130
Steam Turbine, subcritical, 540-MW gross	120
Steam Turbine, supercritical, 860-MW gross	110
Oil-Fired Boiler, subcritical, nominal 300 MW (cursory bid)	200
Gas Turbine (from large simple cycle case), 144 MW	240
Gas Turbine (from large combined cycle case), 191 MW	220
Diesel Engine-Generator, 1.4 MW	290
Diesel Engine-Generator, 4.8 MW	450



Effect of size on equipment cost

$$C_{Y} = C_{W} \left(\frac{X_{Y}}{X_{W}}\right)^{\alpha}$$

 X_Y and X_w are the sizes or capacities of the equipment and C_Y and C_W are the purchase costs of the same type of equipment in the same year but with a different size.



Figure 4.1 Impact of Size on OEM Cost for Simple Cycle Units



net plant output, MW



Figure 4.4 Impact of Size on OEM Costs for Combined Cycle Units

Gas-fired combined cycle units (OEM scope)

(50-Hz units-data from Gas Turbine World Handbook)



net plant output (ISO), MW



Table 5.4 140-MW Combined Cycle Plant–Heavy-Frame Gas Turbine

Each Item Includes Costs for Equipment, Material, and Labor (January 2008 US\$)

Cost Estimate Summary	U.S. (thousands \$)	India (thousands \$)	Romania (thousands \$)
Civil/Structural	7,240	5,130	5,280
Mechanical			
Gas Turbine (OEM Price) ¹	99,740	99,740	99,740
SCR	1,260	630	450
Gas Compressor	2,840	2,790	2,780
Electrical	9,720	8,070	7,590
Piping	9,480	6,680	8,680
Instruments and Controls	1,660	1,510	1,470
Balance of Plant/General Facilities	21,640	14,810	12,830
Total Direct Costs	153,580	139,360	138,820
Indirect Costs	13,490	4,960	3,470
Engineering and Home Office Costs	13,040	5,180	3,840
Process Contingency	0	0	0
Project Contingency	12,060	9,950	9,280
Total Plant Cost	192,170	159,450	155,410
Gas Turbine Cost (FOB-OEM), US\$/kW	730	730	730
Total Plant Cost, US\$/kW	1,410	1,170	1,140



Table 5.5 580-MW Combined Cycle Plan	t-Heavy-Frame Gas	Turbine			
Each Item Includes Costs for Equipment, Material, and Labor (January 2008 US\$)					
Cost Estimate Summary	U.S. (thousands \$)	India (thousands \$)	Romania (thousands \$)		
Civil/Structural	20,120	14,100	14,620		
Mechanical					
Gas Turbine (OEM Price) ¹	262,930	262,930	262,930		
SCR	3,460	1,730	1,230		
Gas Compressor	3,480	3,410	3,390		
Electrical	28,990	24,500	23,180		
Piping	28,190	20,250	26,880		
Instruments and Controls	4,300	3,890	3,760		
Balance of Plant/General Facilities	34,380	30,810			
Total Direct Costs	398,170	365,190	366,800		
Indirect Costs	33,870	12,810	9,210		
Engineering and Home Office Costs	32,750	13,380	10,210		
Process Contingency	0	0	0		
Project Contingency	30,280	25,690	24,660		
Total Plant Cost	495,070	417,070	410,880		
Gas Turbine Cost (FOB-OEM), \$/kW	460	460	460		
Total Plant Cost, \$/kW	860	720	710		



Purchase Equipment Cost (PEC): scaling exponent α

PlantCapacity VariableCapacity RangeExponent αAir (liquid)Product flow rate70–4000 t/d ^a 0.66Air plant (packaged)Compressed0.1–100 Nm³/s0.70dried air ratedried air rate0.1–100 Nm³/s0.70ArgonProduct flow rate450–1800 t/d0.58ArgonProduct flow rate25–500 Nm³/h0.89Carbon blackProduct flow rate45–900 t/d0.67Carbon dioxide (gas orProduct flow rate85–920 t/d0.72liquid)Cogeneration plantNet power5–150 MW0.75Electric power plantNet power1.0–1000 MW0.80
Air (liquid)Product flow rate70-4000 t/d°0.66Air plant (packaged)Compressed0.1-100 Nm³/s0.70dried air ratedried air rate0.1-100 Nm³/s0.70AmmoniaProduct flow rate450-1800 t/d0.58ArgonProduct flow rate25-500 Nm³/h0.89Carbon blackProduct flow rate45-900 t/d0.67Carbon dioxide (gas orProduct flow rate85-920 t/d0.72liquid)Cogeneration plantNet power5-150 MW0.75Electric power plantNet power1.0-1000 MW0.80
Air plant (packaged)Compressed dried air rate0.1-100 Nm³/s0.70AmmoniaProduct flow rate450-1800 t/d0.58ArgonProduct flow rate25-500 Nm³/h0.89Carbon blackProduct flow rate45-900 t/d0.67Carbon dioxide (gas orProduct flow rate85-920 t/d0.72liquid)Cogeneration plantNet power5-150 MW0.75Electric power plantNet power1.0-1000 MW0.80
dried air rateAmmoniaProduct flow rate450–1800 t/d0.58ArgonProduct flow rate25–500 Nm³/h0.89Carbon blackProduct flow rate45–900 t/d0.67Carbon dioxide (gas orProduct flow rate85–920 t/d0.72liquid)Cogeneration plantNet power5–150 MW0.75Electric power plantNet power1.0–1000 MW0.80
AmmoniaProduct flow rate450–1800 t/d0.58ArgonProduct flow rate25–500 Nm³/h0.89Carbon blackProduct flow rate45–900 t/d0.67Carbon dioxide (gas orProduct flow rate85–920 t/d0.72liquid)Cogeneration plantNet power5–150 MW0.75Electric power plantNet power1.0–1000 MW0.80
Argon Product flow rate 25–500 Nm³/h 0.89 Carbon black Product flow rate 45–900 t/d 0.67 Carbon dioxide (gas or Product flow rate 85–920 t/d 0.72 liquid) Cogeneration plant Net power 5–150 MW 0.75 Electric power plant Net power 1.0–1000 MW 0.80
Carbon black Product flow rate 45–900 t/d 0.67 Carbon dioxide (gas or Product flow rate 85–920 t/d 0.72 liquid) Cogeneration plant Net power 5–150 MW 0.75 Electric power plant Net power 1.0–1000 MW 0.80
Carbon dioxide (gas or Product flow rate 85–920 t/d 0.72 liquid) Cogeneration plant Net power 5–150 MW 0.75 Electric power plant Net power 1.0–1000 MW 0.80
liquid)Cogeneration plantNet power5–150 MW0.75Electric power plantNet power1.0–1000 MW0.80
Cogeneration plantNet power5–150 MW0.75Electric power plantNet power1.0–1000 MW0.80
Electric power plant Net power 1.0–1000 MW 0.80
Ethanol Product flow rate $(4-40) \times 10^{\circ} \text{ m}^{\circ}/\text{yr}$ 1.00
$(40-400) \times 10^3 \text{ m}^3/\text{yr}$ 0.90
Liquified natural gas Product flow rate $(1-20) \times 10^3 \text{ t/d} = 0.68$
Methanol Product flow rate $(1.2-18) \times 10^6 \text{ t/yr}^{\prime\prime} = 0.78$
Natural gas purification Product flow rate 18-270 t/d 0.75
Nitrogen (liquid) Product flow rate 70-4000 t/d 0.66
Oxygen (gaseous) Product flow rate 35-900 t/d 0.59
Oxygen (liquid) Product flow rate 500-2700 t/d 0.37
Refinery (complete) Feed flow rate $(9-120) \times 10^3 \text{ bbl/d}^{\alpha} = 0.86$
Refrigeration unit Cooling load 0.05-10 MW 0.70
Refuse-to-electricity Net power 10-150 MW 0.75
Sour gas treating Feed flow rate 0.5-16 Nm ³ /d 0.84
Sulfuric acid Product flow rate 85-1000 t/d 0.56
SNG from coal Product flow rate $(0.5-5.5) \times 10^6 \text{ Nm}^3/\text{d}$ 0.75
Wastewater treatment plant Water flow rate 0.005-5 m ³ /s 0.67
Water desalination Water flow rate 0.05-3 m ³ /s 0.89

"The symbols t/d and t/yr refer to metric tons per day and metric tons per year, respectively. The symbol bbl/d means barrels per day.



Cost Indices

All cost data used in an economic analysis must be brought to the same reference year: the year used as a basis for the cost calculations. For cost data based on conditions at a different time, this is done with the air of an appropriate *cost index*.

 C_{new} can be obtained from reference year cost data C_{ref} by using appropriate cost indices I_{new} and I_{ref} of the year of the required estimate and the reference year

$$C_{new} = C_{ref} \left(\frac{I_{new}}{I_{ref}} \right)$$

Cost indices are frequently published in several journals



Roosen Cost Functions (CCGTs; 2002)

$$\begin{split} C_{comp} &= c_{11} \cdot \dot{m}_{air} \cdot \frac{1}{c_{12} - \eta_{ss,comp}} \cdot \beta \cdot \ln(\beta) \\ C_{cc} &= c_{21} \cdot \dot{m}_{air} \cdot (1 + e^{c_{22} \cdot (T_{out} - c_{23})}) \cdot \frac{1}{0.995 - (\frac{p_{out}}{p_{in}})} \\ C_{turb} &= c_{31} \cdot \dot{m}_{gas} \cdot \frac{1}{c_{32} - \eta_{ss,turb}} \cdot (1 + e^{c_{33} \cdot (T_{in} - 1570 \ K)}) \\ C_{sT} &= c_{51} \cdot \dot{W}_{5T}^{0.7} \cdot \left(1 + (\frac{0.05}{1 - \eta_{ss,ST}})^3\right) (1 + 5 \cdot e^{(\frac{T_{in} - 866}{10.42})}) \\ C_{HRSG} &= c_{41} \sum_{i} f_{p,i} f_{\tau,w,i} f_{\tau,g,i} (\frac{\dot{Q}_{i}}{dT_{in,i}})^{0.8} + c_{42} \sum_{j} f_{p,j} \dot{m}_{steam,j} + c_{43} \dot{m}_{g}^{12} \\ C_{cT} &= c_{61} \frac{\dot{Q}_{cT}}{k \cdot \Delta T_{in}} + c_{62} \dot{m}_{cw} + 70.5 \dot{Q}_{cT} (-0.69 \ln(\bar{T}_{cw} - T_{wb}) + 2.1898) \\ C_{pump} &= c_{71} \cdot \dot{W}_{pump}^{0.71} \cdot \left(1 + \frac{0.2}{1 - \eta_{ss,pump}}\right) \end{split}$$

 $C_{GT,gen}=3082\cdot\dot{W}_{GT}^{0.58}$



Roosen Cost Functions (CCGTs; 2002)

Air compressor:

$$C_{AC} = c_{11} \cdot \dot{m}_{air} \cdot \frac{1}{c_{12} - \eta_{5C}} \cdot \Pi_C \cdot \ln(\Pi_C)$$

$$c_{11} = 44.71 \ \$ \cdot (kg \cdot s)^{-1}$$

$$c_{12} = 0.95$$

Combustion chamber:

$$C_{CC} = c_{21} \cdot \dot{m}_{air} \cdot \left(1 + e^{c_{22} \cdot (T_{out} - c_{23})}\right) \cdot \frac{1}{0.995 - p_{out}/p_{in}}$$

$$c_{21} = 28.98 \ (kg \cdot s)^{-1}$$

$$c_{22} = 0.015 \ K^{-1}$$

$$c_{23} = 1540 \ K$$

Gas turbine:

$$C_{GT} = c_{31} \cdot \dot{m}_{gas} \cdot \frac{1}{c_{32} - \eta_{sT}} \cdot \ln\left(\frac{p_{in}}{p_{out}}\right)$$
$$\times (1 + e^{c_{33} \cdot (T_{in} - 1570 \text{ K})})$$
$$c_{31} = 301.45 \$ \cdot (\text{kg} \cdot \text{s})^{-1}$$
$$c_{32} = 0.94$$
$$c_{33} = 0.025 \text{ K}^{-1}$$

Heat recovery steam generator:

$$C_{\text{HRSG}} = c_{41} \cdot \sum_{i} \left(f_{p,i} \cdot f_{T,\text{steam},i} \cdot f_{T,\text{gas},i} \cdot \left(\frac{\dot{Q}_i}{\Delta T_{\ln,i}} \right)^{0.8} \right) \\ + c_{42} \cdot \sum_{j} f_{p,j} \cdot \dot{m}_{\text{steam},j} + c_{43} \cdot \dot{m}_{\text{gas}}^{1.2}$$

$$\begin{split} f_{p,i} &= 0.0971 \cdot \frac{p_i}{30 \text{ bar}} + 0.9029 \\ f_{T,\text{steam},i} &= 1 + \exp\left(\frac{T_{\text{out},\text{steam},i} - 830 \text{ K}}{500 \text{ K}}\right) \\ f_{T,\text{gas},i} &= 1 + \exp\left(\frac{T_{\text{out},\text{gas},i} - 990 \text{ K}}{500 \text{ K}}\right) \\ c_{41} &= 4131.8 \$ \cdot (\text{kW} \cdot \text{K})^{0.8} \\ c_{42} &= 13380 \$ \cdot (\text{kg} \cdot \text{s})^{-1} \\ c_{43} &= 1489.7 \$ \cdot (\text{kg} \cdot \text{s})^{-1.2} \end{split}$$

Steam turbine:

$$C_{\text{ST}} = c_{51} \cdot P_{\text{ST}}^{0.7} \left(1 + \left(\frac{0.05}{1 - \eta_{s} \text{ST}} \right)^3 \right) \\ \times \left(1 + 5 \cdot \exp\left(\frac{T_{\text{in}} - 866 \text{ K}}{10.42 \text{ K}} \right) \right) \\ c_{51} = 3880.5 \$ \cdot \text{kW}^{-0.7}$$

Condenser and cooling tower:

$$C_{C} = c_{61} \cdot \frac{\dot{Q}_{\text{cond}}}{k \cdot \Delta T_{\text{in}}} + c_{62} \cdot \dot{m}_{\text{CW}} + 70.5 \cdot \dot{Q}_{\text{cond}}$$

$$\times (-0.6936 \cdot \ln(\overline{T}_{\text{CW}} - T_{\text{WB}}) + 2.1898)$$

$$c_{61} = 280.74 \text{ } \cdot \text{m}^{-2}$$

$$c_{62} = 746 \text{ } \cdot (\text{kg} \cdot \text{s})^{-1}$$

$$k = 2200 \text{ } \text{W} \cdot (\text{m}^{2} \cdot \text{K})^{-1}$$

Feed water pump:

$$C_{p} = c_{71} \cdot P_{p}^{0.71} \left(1 + \frac{0.2}{1 - \eta_{s} p} \right)$$
$$c_{71} = 705.48 \, (\text{kg·s})^{-1}$$



Di seguito sono riportati alcuni esempi per i componenti di una microturbina

COMPRESSORE

ł

$$C_{AC} = \frac{c_{11} \cdot m_a}{0.9 - \eta_c} \beta \cdot \ln \beta \qquad \eta_c < 0.9$$

COMBUSTORE

 $C_{CC} = c_{21} \cdot m_a \cdot \left(1 + e^{c_{23} \cdot T_{comb} - c_{24}}\right)$

TURBINA

$$C_{AC} = \frac{c_{31} \cdot m_g}{0.92 - \eta_t} \left(1 + e^{c_{33} \cdot T I T - c_{34}} \right) \ln \beta \qquad \eta_t < 0.92$$

Similar to Roosen (but different constants!)



Credits: Politecnico di Torino, Corso Prof. V. Verda (2013)

SCAMBIATORE DI CALORE

 $C_{P\!H}=c_{41}\cdot A^{0.6}$

HRSG

 $C_{H\!R\!S\!G} = c_{51} \cdot A^{0.6}$

C11=79 €/(kg/s) C21=256 €/(kg/s) C23=0.018 1/°C C24=14 C31=532 €/(kg/s) C33=0.036 1/°C C34=54.4 C41=2120 €/m^{0.6} C51=3250 €/m^{0.6}



$$\begin{split} Z_{1} &= \zeta_{11} y_{1}^{b} [g_{p}g_{1q}g_{1T}, \qquad Z_{2} &= \zeta_{21} y_{d}^{b} [g_{2q}g_{2T}, \qquad Z_{1} &= \beta_{0} [der \\ Z_{3} &= T_{2} (\zeta_{31} n_{3}R_{3} + \zeta_{32} / c_{ps}) y_{3,1} / [T_{0} (T_{b} - T_{a})], \\ Z_{4} &= \zeta_{41} y_{4}^{b} [f_{4m}g_{4m}, \qquad \Gamma_{0,k} = c_{k} y_{0,k} \quad (k = 1, 2, 3), \qquad Z_{2} - ST \\ A &= \zeta_{41} y_{4}^{b} [f_{4m}g_{4m}, \qquad \Gamma_{0,k} = c_{k} y_{0,k} \quad (k = 1, 2, 3), \qquad Z_{3} = c_{0} \log a \\ A &= \zeta_{41} (C/3.6 \times 10^{5} N) \phi_{r} b_{rl}, \qquad g_{p} = \exp[(P_{1} - P_{1}) / b_{13}], \qquad Z_{4} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rl}, \qquad g_{p} = \exp[(P_{1} - P_{1}) / b_{13}], \qquad Z_{4} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rl}, \qquad g_{p} = \exp[(P_{1} - P_{1}) / b_{13}], \qquad Z_{4} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rl}, \qquad g_{p} = \exp[(P_{1} - P_{1}) / b_{13}], \qquad Z_{4} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rl}, \qquad Z_{5} = c_{0} \log a \\ g_{1q} = 1 + [(0, 45 - \bar{\eta}_{0,1}) / (0, 45 - \eta_{0})]^{b_{1,4}}, \qquad The firsts \\ g_{rT} = 1 + b_{r,5} \exp[(T_{1} - \bar{T}_{1}) / b_{rd}] (r = 1, 2), \qquad The firsts \\ g_{rg} = 1 + [(1 - \bar{\eta}_{0,b}) / (1 - \eta_{0,r})]^{b_{r,2}} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rd} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rd} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rd} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rd} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rd} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{rd} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} b_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\ z_{1} = (C/3.6 \times 10^{5} N) \phi_{r} (r = 2, 4), \qquad Z_{5} = c_{0} = 0 \\$$

Energy Vol. 12, No. 7, pp. 563-571, 1987 Printed in Great Britain

0360-5442.87 \$3.00 +0.00 Pergamon Journals Ltd

- construser - pump

THERMO-ECONOMIC FUNCTIONAL ANALYSIS AND OPTIMIZATION

CHRISTOS A. FRANGOPOULOS



Attala Cost Functions (CCGTs; 2001)

Component	Function
(1.2) Multi-level HRSG	$C_{\rm HRSG} = 17000 \left[\sum_{i=1}^{n} \left(\frac{\dot{\underline{\rho}}_{\rm econ}}{\Delta T_{\rm econ}} \right)_{i}^{0.6} + \sum_{i=1}^{n} \left(\frac{\dot{\underline{\rho}}_{\rm evap}}{\Delta T_{\rm evap}} \right)_{i}^{0.6} + \sum_{i=1}^{2} \left(\frac{\dot{\underline{\rho}}_{\rm sh}}{\Delta T_{\rm sh}} \right)_{i}^{0.6} + \left(\frac{\dot{\underline{\rho}}_{\rm LTE}}{\Delta T_{\rm LTE}} \right)^{0.79} \right]$
(2) Gas turbine	$C_{\text{TAG}} = 3832W^{0.71}$
(3) Steam turbine	$C_{\rm TAV} = 3197280A^{0.261} + 823.7W^{1.543}$
(4.1) Titanium condenser	$C_{\rm Cond} = 17769S^{0.516}$
(4.2) Copper-nickel con-	$C_{\rm Cond} = 2296S^{0.79}$
denser	
(4.3) Alluminium-brass	$C_{\rm Cond} = 162S^{1.01}$
condenser	
(4.4) Stainless condenser	$C_{\rm Cond} = 1.7(162S_{\rm eq}^{1.01})$
(5.1) Condenser pump	$C_{\text{P.Cond}} = 37.6W^{0.8} \left[1 + \left((1 - 0.7) / (1 - \eta_{\text{iso}}) \right)^{-0.46} \right] 34.4$
(6) Generator	$C_{\rm ALT} = 3082W^{0.58}$





PERGAMON

Energy Conversion and Management 42 (2001) 2163-2172

www.elsevier.com/locate/enconman

Thermoeconomic optimization method as design tool in gassteam combined plant realization

L. Attala, B. Facchini, G. Ferrara*

Dipartimento di Energetica, "S: Stecco", Università degli Studi di Firenze, Via S. Marta, 3, 50139 Florence, Italy



Turbina a Gas:

$$C_{GT} = \begin{cases} 6380.5 \cdot W_{GT}^{0.73}, W_{GT} < 50MW \\ 3968.8 \cdot W_{GT}^{0.78}, W_{GT} > 50MW \end{cases}$$

La potenza è espressa in [kW].

Turbina a vapore:

$$C_{ST} = 5075.5 \cdot W_{ST}^{0.70} \cdot \left[1 + \left(\frac{0.05}{1 - \eta_{ST}}\right)^3 \right] \cdot \left[1 + 5 \cdot exp\left(\frac{T_{ST,in} - 866K}{10.42K}\right) \right]$$

Condensatore:

$$C_{cond} = 248 \cdot A_{cond} + 659 \cdot m_{steam}$$

Carcasci, C., Facchini, B., Esercitazioni di Sistemi Energetici, Esculapio, 2016 Pompa:

$$C_{pump} = 940 \cdot W_{pump}^{0.71} \cdot \left[1 + \left(\frac{0.20}{1 - \eta_{pump}} \right) \right]$$

Alternatore:

 $C_{alt} = 4028.1 \cdot W_{alt}^{0.58}$



Carcasci Cost Functions (CCGTs; 2016)

Caldaia a Recupero:

$$C_{HRSG} = 5404.2 \cdot \sum_{i} \left\{ f_{p,i} \cdot f_{T,steam,i} \cdot f_{T,gas,j} \cdot UA_i^{0.8} \right\} + 17500.2 \cdot \sum_{i} \left\{ f_{p,i} \cdot m_{steam,i} \right\} + 1948.4 \cdot m_{gas}^{1.2}$$

$$f_{p,i} = 0.0971 \cdot \frac{P_i}{30bar} + 0.9029$$

$$f_{T,steam,i} = 1 + exp\left(\frac{T_{steam,out,i} - 830K}{500K}\right)$$

$$f_{T,gas,i} = 1 + exp\left(\frac{T_{gas,in,i} - 990K}{500K}\right)$$

$$UA_i = \frac{Q_i}{\Delta T_{ml}} \left[\frac{kW}{K}\right]$$

/K

(Same as Roosen, 2003...?)



Air compressor

$$Z_{C} = \left(\frac{c_{11}\dot{m}_{air}}{c_{12} - \eta_{is,C}}\right) \left(\frac{P_{out}}{P_{in}}\right) \ln \left(\frac{P_{out}}{P_{in}}\right)$$
$$c_{11} = 75 / (kg/s), c_{12} = 0.9$$

Combustion and post combustion chamber

 $Z_{CC} = c_{21} \cdot \dot{m}_{air(gas)} \cdot (1 + exp(c_{22}T_{out} - c_{23})) \cdot \frac{1}{0.995 - \frac{P_{out}}{P_{in}}}$ $c_{21} = 48.64 / (kg/s), c_{22} = 0.018 \text{ K}^{-1}, c_{23} = 26.4$

Gas turbine

$$Z_{\rm GT} = \left(\frac{c_{31} \dot{m}_{\rm gas}}{c_{32} - \eta_{\rm is,GT}}\right) \left(\frac{P_{\rm out}}{P_{\rm in}}\right) (1 + \exp(c_{33}T_{\rm in} - c_{34}))$$

$$c_{31} = 1536 / (\rm kg/s), c_{32} = 0.92, c_{33} = 0.036 \ \rm K^{-1}$$

HRSG

$$\begin{aligned} Z_{\text{HRSG}} &= c_{41} \cdot \sum_{i} \left(f_{p,i} \cdot f_{T.\text{steam},i} \cdot f_{T.\text{gas},i} \cdot \left(\frac{\dot{Q}_{i}}{\text{IMTD}_{i}} \right) \right. \\ &+ c_{42} \cdot \sum_{j} f_{p,j} \cdot \dot{m}_{\text{steam},j} + c_{43} \cdot \dot{m}_{\text{gas}}^{1.2} \right. \\ f_{p,i} &= 0.0971 \cdot \frac{p_{i}}{30 \text{ bar}} + 0.9029 \\ f_{T.\text{steam},i} &= 1 + \exp\left(\frac{T_{\text{out},\text{steam},i} - 830 \text{ K}}{500 \text{ K}} \right) \\ f_{T.\text{gas},i} &= 1 + \exp\left(\frac{T_{\text{out},\text{steam},i} - 830 \text{ K}}{500 \text{ K}} \right) \\ f_{41} &= 4131.8 \quad (\text{kW K})^{0.8} \\ c_{42} &= 13380 \quad (\text{kg s})^{-1} \\ c_{43} &= 1489.7 \quad (\text{kg s})^{-1.2} \end{aligned}$$



Energy Conversion and Management 76 (2013) 83-91

A comparative exergoeconomic analysis of two biomass and co-firing combined power plants



S. Soltani^a, S.M.S. Mahmoudi^{a,*}, M. Yari^b, T. Morosuk^c, M.A. Rosen^d, V. Zare^a



Steam turbine

$$Z_{\text{ST}} = c_{51} \cdot \dot{W}_{\text{ST}}^{0.7} \left(1 + \left(\frac{0.05}{1 - \eta_{\text{is,ST}}} \right)^3 \right) \times \left(1 + 5 \cdot \exp\left(\frac{T_{\text{in}} - 866 \text{ K}}{10.42 \text{ K}} \right) \right)$$

$$c_{51} = 3880.5 \text{ kW}^{-0.7}$$

Condenser and cooling tower

$$Z_{\text{Cond}} = c_{61} \cdot \frac{\dot{Q}_{\text{Cond}}}{2.2.1\text{MTD}} + c_{62} \cdot \dot{m}_{\text{CW}} + 70.5 \cdot \dot{Q}_{\text{Cond}}$$
$$\times (-0.6936 \cdot \ln(\bar{T}_{\text{CW}} - T_{\text{WB}}) + 2.1898)$$
$$c_{61} = 280.74 \text{ m}^{-2}$$
$$c_{62} = 746 \text{ (kg s)}^{-1}$$

Pump

$$Z_{\text{pump}} = c_{71} \cdot \dot{W}_{\text{pump}}^{0.71} \left(1 + \frac{0.2}{1 - \mu_{\text{is pump}}} \right)$$
$$c_{71} = 705.48 \ (\text{kg s})^{-1}$$

Air preheater

$$Z_{AP} = c_{81} \cdot A_{AP}^{0.6}$$

 $c_{81} = 4122$
 $U = 0.018 \text{ kw}/(\text{m}^2 \text{ K})$
Gasifier

 $Z_{\text{gasifier}} = 1600 \cdot \left(\dot{m}_{\text{dry-biomass}} [\text{kg/h}] \right)^{0.67}$

Soltani et al. Cost Functions (CCGTs; 2013)



Air compressor :

$$Z_{C} = \left(\frac{C_{11}}{C_{12} - \eta_{sc}}\right) r_{p} \ln(r_{p})$$
$$C_{11} = 71.1 \ \$ / (kgs^{-1}) \quad C_{12} = 0.9$$

$$Z_{CC} = \left(\frac{C_{21}\dot{m}_a}{C_{22} - 0.98}\right)$$

 \boldsymbol{c}

Combustion chamber :

 $\times (1 + \exp(C_{32}T_{comb} - C_{24}))$

 $C_{21} = 46.08, C_{22} = 0.995,$ $C_{22} = 0.018, C_{24} = 26.4$

Gasturbine :

 $Z_{GT} = \left(\frac{C_{31}\dot{m}_g}{C_{32} - \eta_{GT}}\right) \ln\left(\frac{P_4}{P_3}\right) (1 + \exp(C_{33}T_3 - C_{34}))$ $C_{31} = 479.34, C_{32} = 0.92, C_{33} = 0.036, C_{34} = 54.4$

Applied Thermal Engineering 91 (2015) 848-859



Research paper

Exergoeconomic multi-objective optimization of an externally fired gas turbine integrated with a biomass gasifier



Shoaib Khanmohammadi^{*}, Kazem Atashkari, Ramin Kouhikamali





degli studi FIRENZE

Air preheater : $Z_{AP} = C_{41} \left(\frac{\dot{m}_5 (h_5 - h_6)}{U \Delta T_{IM}} \right)^{0.5}$ $U = 6, C_{41} = 4122$ Gasifier: $Z_G = 1600(3600 \times \dot{m}_{biomass})^{0.67}$ Domestic hot water heater : $Z_{DHW} = 0.3 \dot{m}_{DHW}$ ORC evaporator : $Z_{Ev,R} = 309.14(A_{Ev})^{0.85}$ $ORC \ pump: \quad Z_{Pump,R} = 200 \Big(\dot{W}_{Pump} \Big)^{0.65}$ $ORC \ turbine: \ \ Z_{Tur,R} = 4750 \Big(\dot{W}_{tur} \Big)^{0.75}$ ORC condenser : $Z_{Cond.R} = 516.62(A_{Condnser})^{0.6}$



Compressor	$I_{\rm C} = c_1 39.5 \dot{G}_{\rm a} \pi_{\rm C} \ln(\pi_{\rm C}), \text{USD}$	(11)
Combustion Chamber	$I_{\text{COMB}} = c_1 25.6 \dot{G}_{\text{fg}} [1 + \exp(0.018T_{\text{COMB}} - 26.4c_2)], \text{USD}$	(12)
Gas Turbine $I_E = c_1 266.3 \dot{G}_{fg} \ln(\pi_E) [1 + exp(0.036T_{COMB} - 54.4c_2)], USD$		
	while	
	$c_1 = 21; c_2 = 1.207$	(14)
HRSG	$I_{\rm HRSG} = 21200 \left[\sum_{n} \left(\frac{\dot{Q}}{\Delta T_{\rm log}} \right)_{n}^{0.6} + \sum_{m} \left(\frac{\dot{Q}}{\Delta T_{\rm log}} \right)_{m}^{0.79} \right], \rm USD$	(15)
Steam Turbine	$I_{\rm ST} = 1.1 (3197280 A_{\rm ST}^{0.261} + 823.7 N_{\rm ST}^{1.543}), \text{USD}$	(16)
Condenser	$I_{\text{COND}} = 2870A_{\text{COND}}^{0.79}, \text{USD}$	(17)
Con Energy Co	tents lists available at ScienceDirect Driversion and Management age: www.elsevier.com/locate/enconman	

Economic optimization of the combined cycle integrated with multi-product gasification system

M. Liszka*, A. Ziebik



Cost function of the various pieces of equipment.

Equipment	Average [kUSD 2009]	Optimistic [kUSD 2009]	Pessimistic [kUSD 2009]	Variable	Max, size of a unit
Turbine Compressor Pump HX Cold storage Press, vessel	$2 X^{0.6} + 40$ $9 X^{0.6} + 20$ $44 X^{0.9} + 31$ $0.45 X^{0.82} + 5$ $15 X$ $4.2 X^{0.6} + 10$ $0.5 X^{0.785} + 0$	$\frac{1.5 X^{0.6} + 10}{6 X^{0.6} + 10}$ $\frac{44 X^{0.75} + 20}{0.3 X^{0.82} + 1}$ $\frac{10 X}{2 X^{0.6} + 2}$	$\begin{array}{r} 4 X^{0.6} + 100 \\ 10 X^{0.6} + 200 \\ 50 X + 40 \\ 0.8 X^{0.82} + 10 \\ 20 X \\ 7 X^{0.6} + 25 \\ 7 X^{0.785} + 25 \end{array}$	Power [kW] Power [kW] Flow [m ³ s ⁻¹] Area [m ²] Capacity [kWh] Volume [m ³]	 100000 5000
Non-glass SC Glassed SC MDAC	0.33 X ^{0.9} 0.425 X ^{0.9} 1.412 X ^{0.8}	0.2 X + 2 0.355 X ^{0.9} 0.52 X ^{0.9} 0.941 X ^{0.8}	0.355 X 0.52 X 2.683 X ^{0.8}	Area [m ²] Area [m ²] Area [m ²]	- - 2000

Energy 45 (2012) 358-365

	Contents lists available at SciVerse ScienceDirect	ENERGY
	Energy	
ELSEVIER	journal homepage: www.elsevier.com/locate/energy	Construction Construction

Roundtrip Efficiency 57%

Thermoeconomic analysis of a solar enhanced energy storage concept based on thermodynamic cycles

Samuel Henchoz^{a,*}, Florian Buchter^b, Daniel Favrat^a, Matteo Morandin^a, Mehmet Mercangöz^b

^aIndustrial Energy Systems Laboratory, Ecole Polytechnique Fédérale de Lausanne (EPFL), Station 9, CH-1015 Lausanne, Switzerland ^bABB Switzerland Ltd., Corporate Research, Segelhofstrasse 1k, CH-5405 Baden-Dättwil, Switzerland

Special Equipment! TransCrit CO2 power cycle....

Estimated Cost: 1200 €/kW



Per quanto riguarda le altre componenti che costituiscono i costi diretti (DC), qualora non possano essere valutate singolarmente attraverso una dettagliata analisi, è possibile calcolarle come percentuale del costo di acquisto delle macchine (PEC). Si verificano le seguenti condizioni:

Installazione:	20-90 % del PEC
Piping:	10-70 % del PEC
Strumentazione:	6-40 % del PEC
Apparecchiature elettriche:	10-15 % del PEC
Terreno:	10 % del PEC
Opere civili:	10-80 % del PEC
Service facilities:	30-100 % del PEC

In assenza di altre indicazioni è possibile assumere il valore intermedio di ciascun intervallo.

Si sottolinea il fatto che queste valutazioni consentono di definire un ordine di grandezza del costo complessivo. L'errore commesso può essere rilevante in assenza di informazioni più dettagliate.

Credits: Politecnico di Torino, Corso Prof. V. Verda (2013)



Per i costi indiretti e i costi addizionali si può procedere in modo simile a quanto visto per i costi diretti:

Progettazione: Costruzione: Avviamento: Working capital: 25-75 % del PEC 15 % del DC 5-12 % del FCI 10-20 % del TCI

 $FCI = DC + IC \quad (Recursive)$ TCI = FCI + AC = DC + IC + AC (Recursive)



E' possibile utilizzare le informazioni riportate, unitamente ad alcune valutazioni di massima delle altri componenti per fornire una relazione tra il costo di acquisizione dei macchinari (PEC) e il capitale totale associato all'investimento:





Installation Cost

The *installation cost* covers the freight and insurance for the transportation from the factory, the cost for labor, unloading, handling, foundations, supports, and all other construction expenses related directly to the erection and necessary connections of the purchased equipment.

This cost is needed only when the economic analysis is conducted separately for single equipment items or small groups of items.

> Credits: G. Tsatsaronis, TUB, Inspire Summer Course, 2007, Nova Gorica



Estimation of Direct Costs - 1

Piping

The cost for piping includes the material and labor costs of all items required to complete the erection of all the piping used directly in the system.

Instrumentation and Controls

The factor used to estimate these costs tends to increase as the degree of automation increases, and to decrease with increasing total cost.

Electrical Equipment and Materials

This cost includes materials and installation labor for substations, distribution lines, switch gears, control centers, emergency power supplies, area lighting, etc.



Estimation of Direct Costs - 2

Land

The cost of land strongly depends on the location and usually does not decrease with time.

Civil, Structural, and Architecture Work

This category includes the total cost for buildings, including services, as well as the costs for roads, side-walks, fencing, landscaping, yard improvements, etc.

Service Facilities

This cost includes all costs for supplying the general utilities required to operate the system such as fuel(s), water, steam, and electricity (if it is not a product of the system), refrigeration, inert gas, and sewage, waste disposal, environmental control, fire protection, and the equipment required for shops, first aid, and cafeteria.



Indirect Costs

Engineering and Supervision:

Cost for developing the detailed plan design and drawings, and the costs associated with cost engineering, scale models, purchasing, engineering supervision and inspection, administration, travel, and consultant fees.

Construction:

All expenses for temporary facilities and operations, tools and equipment, home office personnel located at the construction site, insurance, etc.

Contingencies:

Estimates are based on assumptions for cost and productivity, which may vary significantly from the actual values. The contingency factor depends on the complexity, size, and uniqueness of the energy conversion system.



Startup Costs

Startup Costs are mainly associated with design changes that have to be made after completion of construction but before the system can operate at design conditions.

The startup costs include

labor, materials, equipment, and overhead expenses to be used only during startup time

plus

the loss on income while the system is not operating or operating at only partial capacity during the same period.

Table 2.1 Historical Average Annual Compound Escalation

D. Pauschert, ESMAP Technical Paper 122/09

Cost Indexing

università degli studi FIRENZE

DIPARTIMENTO DI INGEGNERIA INDUSTRIALE

Study of Equipment Prices in the Power Sector

Ranking	Plant Equipment and Materials	Jan. 1996- Dec. 2003, % per year	Jan. 2004- Dec. 2007, % per year	Jan. 2004- Dec. 2007, % Increase for Period
	United States			
4	Ready-Mix Concrete	1.9	7.9	36
	Centrifugal Pumps	2.0	4.7	20
	Centrifugal Fans	1.7	4.2	18
	Material Handling Conveyors	1.7	4.7	20
	Pneumatic Conveyors	1.7	3.8	16
	Crushers and Pulverizers	2.9	4.4	19
	Integral Horsepower Motors	0.4	6.4	28
	Fabricated Steel Plates	0.3	10.1	47
2	Structural Steel	0.9	8.0	36
	Steel Pipe and Tubing	NA	7.0	31
	Field Erected Steel Tanks	1.5	5.8	25
3	Heat Exchangers and Condensers	0.8	7.8	35
	Fin Tube Heat Exchangers	1.3	8.4	38
	Industrial Mineral Wool	0.4	3.7	16
	Refractory, Non-Clay	0.4	3.7	16
1	Electric Wire and Cable	1.1	9.1	42
	Power and Distribution Transformers	NA	13.8	68
	Copper Wire and Cable	-0.8	18.7	98
	Industrial Process Control Instruments	NA	3.0	12
	India			
	Fabricated Metal (Structural Steel and Plate)	NA	7	31
	Steel Pipe and Tubing	NA	6	26
	Mechanical Equipment	NA	6	26
	Electric Wire and Cable	NA	20	107
	Electric Equipment	NA	7	31
	Romania			
	Fabricated Metal (Structural Steel and Plate)	NA	7	31
	Steel Pipe and Tubing	NA	5	33
	Mechanical Equipment	NA	3	13



Table ES	Historical Average Annual Compound Escalation				
Ranking	Plant Equipment and Materials	Jan. 1996-Dec. 2003, % per year	Jan. 2004-Dec. 2007, % per year		
	United States				
	Fabricated Steel Plates	0.3	10.1		
	Steel Pipe and Tubing	NA	7.0		
	Centrifugal Pumps	2.0	4.7		
	Copper Wire and Cable	-0.8	18.7		
	Power and Distribution Transformers	NA	13.8		
	India				
	Fabricated Metal (Structural Steel/Plate)	NA	7		
	Steel Pipe and Tubing	NA	6		
	Mechanical Equipment	NA	6		
	Electric Wire and Cable	NA	20		
	Electric Equipment	NA	7		
	Romania				
	Fabricated Metal (Structural Steel/Plate)	NA	7		
	Steel Pipe and Tubing	NA	5		
	Mechanical Equipment	NA	3		



Cost Escalation; Projections, Basic Equipment.

Pauschert, ESMAP 2009

Pricing Estimates for Cost of Power Plant Technology, 2008

Table ES2 Projected Future Average Annual Compound Esc	alation
Plant Equipment and Materials	Projected, 2008-2012, % per year
United States	
Fabricated Steel Plates	0 to 2
Structural Steel	2 to 3
Steel Pipe and Tubing	2 to 4
Centrifugal Fans	1 to 3
Electric Wire and Cable	-1 to 2
Power and Distribution Transformers	1 to 3
India	
Fabricated Metal (Structural Steel and Plate)	6 to 8
Steel Pipe and Tubing	8 to 9
Mechanical Equipment	3 to 4
Electric Wire and Cable	1 to 3
Electric Equipment	2 to 4
Romania	
Fabricated Metal (Structural Steel and Plate)	2 to 3
Mechanical Equipment	2 to 3
Steel Pipe and Tubing	2 to 4

Table ES3	Class 5 Pricing Estimates for Selected Generation Technologies (2008 US\$), US\$/kW net					
Generation Plant-Total Plant Cost		U.S.	India	Romania		
Gas Turbine	Combined Cycle Plant, 140 MW	1,410	1,170	1,140		
Gas Turbine	Simple Cycle Plant, 580 MW	860	720	710		
Coal-Fired S	iteam Plant (sub), 300 MW net	2,730	1,690	2,920		
Coal-Fired S	iteam Plant (sub), 500 MW net	2,290	1,440	2,530		
Coal-Fired S	iteam Plant (super), 800 MW net	1,960	1,290	2,250		
Oil-Fired Ste	eam Plant (sub), 300 MW net	1,540	1,180	1,420		
Gas-Fired St	team Plant (sub), 300 MW net	1,360	1,040	1,110		
Diesel Engin	e-Generator Plant, 1 MW	540	470	490		
Diesel Engin	e-Generator Plant, 5 MW	630	590	600		



Table 3.1	Average Annual Compound Escalation for Plant Equipment and Materials-United States				
Figure Number	Equipment or Material Item	Jan. 1996- Dec. 2003, %/year	Jan. 2004- Dec. 2007, %/year	Projected, 2008-2012, %/year	
1	Ready-Mix Concrete	1.9	7.9	2 to 4	
2	Centrifugal Pumps	2.0	4.7	2 to 3	
3	Centrifugal Fans	1.7	4.2	1 to 3	
4	Material Handling Conveyors	1.7	4.7	1 to 2	
5	Pneumatic Conveyors	1.7	3.8	NA	
6	Crushers and Pulverizers	2.9	4.4	NA	
7	Integral Horsepower Motors	0.4	6.4	NA	
8	Fabricated Steel Plates	0.3	10.1	0 to 2	
9	Structural Steel	0.9	8.0	1 to 3	
10	Steel Pipe and Tubing	NA	7.0	2 to 4	
11	Field Erected Steel Tanks	1.5	5.8	NA	
12	Heat Exchangers and Condensers	0.8	7.8	NA	
13	Fin Tube Heat Exchangers	1.3	8.4	NA	
14	Industrial Mineral Wool	0.4	3.7	NA	
15	Refractory, Non-Clay	0.4	3.7	NA	
16	Power and Distribution Transformers	NA	13.8	1 to 3	
17	Electric Wire and Cable	1.1	9.1	-1 to 2	
18	Copper Wire and Cable	-0.8	18.7	NA	
19	Industrial Process Control Instruments	NA	3.0	NA	



Time Value of Money - 1

If $P \in (present value)$ are deposited in an account earning i_{eff} (effective rate of return) per time period and the interest is compounded at the end of each of *n* time periods, the account will grow to $F \in (future value)$

$$F = P(1 + i_{eff})^n$$

Credits: G. Tsatsaronis, TUB, Inspire Summer Course, 2007, Nova Gorica







Matematica Finanziaria

La formulazione precedente può essere utilizzata per riportare all'istante 0, istante di inizio dell'attività produttiva, movimenti di denaro che avvengono nel periodo precedente (cioè le componenti che costituiscono TCI) oppure nel periodo successivo





Annualità: rate costanti (A), pagate alla fine di ciascun periodo per n periodi. Il capitale, pagato in un'unica soluzione all'istante presente, corrispondente alle n annualità è:



Uniform-series present-worth factor

$$\frac{P}{A} = \frac{\left(1 + i_{eff}\right)^n - 1}{i_{eff}\left(1 + i_{eff}\right)^n}$$



Component cost per unit time

Per il calcolo del rateo di costo Z (in €/s) si procede secondo i passi di seguito indicati:

1) Calcolo del costo C dei componenti

 Calcolo del costo totale di investimento TCI per ciascun componente

3) Calcolo delle annualità A utilizzando la formulazione

$$A_j = TCI_j \frac{i \cdot (1+i)^n}{(1+i)^n - 1}$$

4) Calcolo rateo Z

$$Z_j = \frac{A_j}{h \cdot 3600}$$

Essendo h il numero di ore equivalenti di funzionamento annue



In alternativa, il capitale all'istante n corrispondente alle annualità pagate è:





The Payback Period (= Payout Period) is defined as the length of time required for the cash inflow received from a project to recover the original cash outlays required by initial investment.



All expenses, taking place at beginning, intermediate or annual conditions, must be reduced to a common basis for evaluation of the investment (P, F or A). The same holds for yearly incomes when the plant starts production.



The preceding formulas allow to do that.



Methods using discounted cash flows consider the time value of money and all cash flow streams during the life of a Project:

- Net Present Value Method (NPV)
- Internal Rate of Return Method (IRR)

Net present value method

The following rules apply: accept any project for which the present value is positive; reject any project with negative present value; projects with the highest present value are given the highest preference among various alternatives; if two projects are mutually exclusive, accept the one having the greater present value.

Internal rate of return method

It seeks to avoid the arbitrary choice of an interest rate. It calculates an interest rate, initially unknown, that is internal to the project.



L. Attala, B. Facchini, G. Ferrara, Thermoeconomic optimization method as design tool in gassteam combined plant realization, Energy Conversion and Management, 42, 2001, 2163-2172 Ch. Frangopoulos, Thermoeconomic Functional Analysis and Optimization, Energy, 12, 7, 563-571, 1987

Garrett, D.E., Chemical Engineering Economics, Springer, 1989

S. Khanmohammadi, K. Atashkari, R. Kouhikamali, Exergoeconomic multi-objective optimization of an externally fired gas turbine integrated with a biomass gasifier, Applied Thermal Engineering 91, 2015, 848-859

M. Liszka, A. Ziebik, Economic optimization of the combined cycle integrated with multi-product gasification system, Energy Conversion and Management 50 (2009) 309–318

D. Pauschert, Study of Equipment Prices in the Power Sector, ESMAP Technical Paper 122/09 M.S. Peters, K. Timmerhaus, R.E. West, *Plant Design and Economics for Chemical Engineers*, McGraw-Hill

R. Turton, R.C. Bailie, W.B. Whiting, J.A. Shaeiwitz, *Analysis, Synthesis and Design of Chemical Processes*, Prentice-Hall

P. Roosen, S. Uhlenbruck, K. Lucas, Pareto optimization of a combined cycle power system as a decision support tool for trading off investment vs. operating costs, International Journal of Thermal Sciences, 42, 2003, 553–560

S. Soltani, S.M.S. Mahmoudi, M. Yari, T. Morosuk , M.A. Rosen , V. Zare, A comparative exergoeconomic analysis of two biomass and co-firing combined power plants, Energy Conversion and Management ,76 , 2013, 83–91